

ALSTOM STANDARD FOR RAILWAY APPLICATIONS FASTENED ASSEMBLIES DESIGN





TABLE OF MODIFICATIONS

Revision	Publication	Summary of Changes	
0	01/12/22	Rationalization table, standard nut, design rules section and imperial fastener	
N	24/03/21	New template, update of rationalization table	
М	08/12/17	Rationalization table hexalobular screw / Table washer association rules	
L	12/01/17	Rationalization table / NAV suppression / association rules / Tightening torques values indications	
K	08/04/14	Rationalization table / nut with plastic insert / FS nut, superbolt	
J	10/10/13	Hexalobular screw - update of the NOTA - Dimensional rules	
I	02/07/13	Authorized and forbidden fastener list - Update of the NOTA	
Н	20/01/12	Definitions of secure assemblies	
G	12/06/10	Suppression of solicitation category	
F	10/10/09	Document reformatting following revisions mandated by Central Product Engineering	
E	24/04/07	Performance levels / Preamble/ Galvanic corrosion / Correspondence with DTRF 150213 and assemblies / suppression of theme-oriented organization / tightening torque	
D	06/06/06	Title modification / Special shear constraints 6.2 Elimination of appendix: Friction factor 8.2 Bolted assemblies title block 5.3 Suppression of image of test on tensile binary 5.4.5	
С	04/10/05	Correction anti-binding paste / Quality level C1 clamping: revised explanation / Suppression of chapter 6.2 special groupings following meeting of October 4th Return of experience on Nordlock solution – chapter 5.4.2.2	
В	20/06/05	Correction of table on corrosion resistance, as well as designation example and addition of naming of bolted assemblies. Correction of clamping descriptions according to quality levels. Referencing to std. PrEN 15085 for the definition of security levels Appendix addition: Reminder for designers	
А	01/06/05	Re-reading and addition of theoretical part following CCN meeting on 11/05/05	
/	13/10/04	Original issue	





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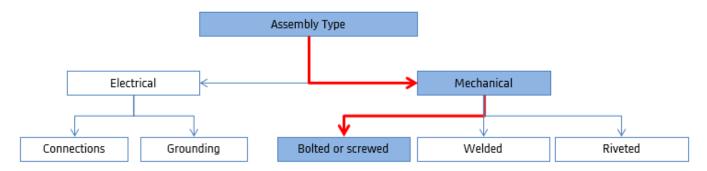
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1. PURPOSE

This ALSTOM Standard deals with railway screwed or bolted mechanical assemblies for structural or non-structural applications, as described below. It does not cover electrical assemblies, purpose of DTRF 150222.



The purpose of this ALSTOM Standard is to define the performance class of an assembly, as well as the fundamental rules for design and dimensioning. This ALSTOM Standard neither aims at defining the standards of screwed elements (screwed elements catalogue), nor at defining procurement requirements (DTRF 150213).

ALSTOM's internal application of the mechanical fastening elements lists (authorized, forbidden, etc.) that appear in this ALSTOM Standard is used for new projects via the application of the standard element's lists: PPL (Preferred Part List).

Application of these technical solutions by suppliers:

- is not mandatory but strongly recommended for their component
- mandatory at the ALSTOM interface and approved during contract review.

In absence of European standard governing railway bolted joints and only when DIN 25201 is authorized for the considered project, this DTRF 150210 complements this standard.

Even if International Standards exist in different languages, English version is considered as official reference.





2. NORMATIVE REFERENCES

2.1. STANDARD REFERENCES

Reference	Title
EN 1993-8	Design of steel structure – Design of joints
ISO 273	Clearance hole for bolts and screws

2.2. ALSTOM STANDARD REFERENCES

Reference	Title	
DTRF 150212	Assemblies and parts criticality management	
DTRF 150213	Fasteners Purchasing Specification	
DTRF 150214	Fastened Assemblies Mounting Requirements	
DTRF 150250	Design Annotation & Dimension Instruction (DADI)	
DTRF 150262 Washing and Cleaning Agents Specification		
DTRF 150611	Surface protection catalogue	





3. TERMS AND DEFINITION

Color codes definition:

Authorized or solution to be promoted

Prohibited solution

Authorized under condition, restricted use

TERM	DEFINITION			
ASSEMBLIES				
Bolted assembly A bolted assembly consists in a screw, at least two parts to be assembled together, a nut possibly one or two washers.				
Screwed assembly	A screwed assembly consists in a screw, a washer and a tapped part. It may also be used more broadly to designate an assembly containing at least one screw.			
Structural assembly	An assembly is structural when its failure directly or indirectly leads to a dangerous situation for passengers, the operator, the maintenance personnel or leads to the damaging of the rolling stock itself. The term structural designates the removable or permanent mechanical connections of the			
Structural assembly	train's sub-assemblies (cabin, side, underframe, roof, cross-members;), as well as equipment (seats, transformers) and force transmission (anti roll bar, damper,) These assemblies are exposed to high static and fatigue loads during normal service of the stock.			
Semi Structural assembly	A semi structural assembly is by definition a non-structural connection subject only to static loads during normal operating service of the rolling stock.			
Fastening assembly	A fastening assembly is not subject to static or fatigue loads during normal operating service of the stock. Is used for lining, interior fitting, signboards, The loosening of the connection does not lead to any dangerous situation for the user. The fastening assemblies are mainly associated to a low safety level.			
Permanent or quasi permanent assembly	Assembly rarely disassembled (twice during rolling stock life), at mid-life (15 years) or in case of accident			
Screw influence diameter: Da	Exterior diameter of disk materializing contact zone between 2 assembled parts and which center is screw axis (Rötscher cone)			
Clamping thickness: Lk	For bolted assemblies, it consists in sum of parts thickness located between screw head bottom and nut bottom (beginning of tapping in case of screwed assembly)			
Tapping or nut implantation length: Li	Engagement length of the threads in an internal thread or length of the screw/nut contact.			
Thread pitch, p	The pitch value is the distance covered in one turn and is measured between two vertices of the thread			





Nominal load or nominal pre- stress inserted: F0	Load or pre-stress inserted into assembly corresponding to average friction coefficient and to average tightening torque			
Maximum load or maximum pre-stress inserted: F0max				
Minimum load or minimum pre-stress inserted: F0min	Load or pre-stress inserted into assembly corresponding to maximum friction coefficient and to minimum tightening torque			
Required limit load: Tr	Limit load to be ensured to fulfill non-slipping and non-separation conditions. Minimum load F0min in the assembly must always be greater than required limit load.			
Definition of the exposed areas	Z _{out} : Outside area (formerly known as Zone C) Z _{inW} : Inside area Wet (formerly known as Zone B) Z _{inD} : Inside area Dry (formerly known as Zone A)			
Loose fit	It's a none tight fitted assembly - Maximum relative movement of the two components depending on the gap between screw diameter and diameter of hole (see also table for size of clearance holes)			
Transition fit or tight fitted Form fit which restricts relative movement of two components (tolerances e.g. High tight fitted assembly.				
ASP	Assembly Surface Protection: insertion of limited paint thicknesses or grease or specific product to optimize friction factor or only to avoid galvanic corrosion at interface			
	THREADED FASTENERS			
Thread forming screw or tapping screw self-threading screws No shawes and resistant thread because the metal is pushed. (ex: used for mecha assemblies: traction cubicle - Tarbes)				
Thread cutting screw	Use for fastening: inconvenient: shavings, risk of crack initiation (ex: used for fastening signboards – Creusot)			
Self-drilling screw	Screw drills the hole before forming the thread: inconvenient: shavings, machining of thread with risk of crack initiation			
	FLEXIBILITY			
Some recommendations are given in chart form and each function is characterized by the limits to be reached. For each of the functions are associated one or several acceptance criteria (requirement requested or necessary for the implementation of the assembly), as well as the levels (quantification of the criterion on an adapted scale) with a flexibility (tolerance).				
Fo	The specified conditions cannot be challenged in any circumstances. The requirement must be met.			
F1	The requirement is not mandatory but must be followed, unless an alternative solution can be demonstrated to be at least equivalent.			

• Size of clearance holes: F, M, L clearance holes for metric dimensions





Screws diameter	Screw clearance holes diameter (ISO 273) for a round or oblong hole		
	F	M	L
M6	6.4	6.6	7
M8	8.4	9	10
M10	10.5	11	12
M12	13	13.5	14.5
M16	17	17.5	18.5
M20	21	22	24
M24	25	26	28

Size of clearance holes: for imperial dimensions

Screw diameter	Screw clearance holes diameter (ASME B18.2.8) for a round or oblong hole		
	Close Fit	Normal Fit	Loose Fit
#4	0.120	0.128	0.144
#6	0.154	0.170	0.185
#8	0.180	0.196	0.213
#10	0.206	0.221	0.238
1/4"	0.266	0.281	0.297
5/16"	0.328	0.344	0.359
3/8"	0.391	0.406	0.422
1/2"	0.531	0.562	0.609
5/8"	0.656	0.688	0.734
3/4"	0.781	0.812	0.906
1"	1.031	1.094	1.156





4. ASSEMBLY CLASSIFICATION

Every railway bolted assembly is defined by a safety level (/H /M /L) and a performance class (CP).

A safety level does not imply a class of performance of tightening. If necessary, industrial department can adapt the tool for tightening according to a safety level while respecting the performance class defined by the engineering office.

4.1. BOLTED ASSEMBLY SAFETY LEVEL

The 3 safety levels (High, Medium or Low – DTRF150212) of an assembly define:

Assembly safety level	High (H)	Medium (M)	Low (L)
Calculation of the assembly and formalization	Mandatory		Optional
Assembly and inspection requirements		DTRF 150214	
Sourcing prescriptions of fastening elements		DTRF 150213	

4.2. BOLTED ASSEMBLY PERFORMANCE CLASS

A screwed assembly is designated by a performance class (CP) defined by the engineering office, determined according to the constraints related to tightening precision required by dimensioning, accessibility of the assembly and during the maintenance.

The 5 performance classes used within ALSTOM define the maximum precision required for the use of a tension or torque applied to an assembly by considering tool precision **as well as human factor:**

- +/-20 % precision tools are not appropriate for a CPC20 performance class.
- Capability (as per DTRF 150214) allows verifying the adequacy of means (sensibility) considering the human factor towards the specified performance class.
- CPC20 torque values considered as a preferred solutions for all designs.

Assembly Performance Class	Tightening method	Maximum precision required	Tightening tool example	Assembly and inspection requirements
СР В	Tension (kN)	< ± 10%	Hydraulic tensioner - Fusible HRC screw	
CP C10	T	< ± 10%	Torque tightening (wrench)	DTRF 150214
CP C20	Torque (N.m)	< ± 20%	, ,	
CP D		NA	Tightening without the use of a torque wrench (flat or impact wrench)	





5. DELIVRABLES

5.1. ASSEMBLY DESCRIPTION

- Indicate the DTR code of the fixation element in the bill of material
- Prescribe salt spray level, in hours, if different from standards DTRF 150213
- Use the logos of the DTRF 150250 to specify:
 - Tightening torque and performance class
 - Safety level (/H / M / L)
 - Safety logo
 - Geometrical tolerances on interfaces (hole, flatness, roughness)
 - Additional protection on fastener (wax, peripherical sealing)

5.2. ADDITIONAL INFORMATION

• Retightening operation with parameter defined with the site assembly expert (ENG/IND): duration + retightening torque (if different from initial torque value):

Example Re-tightening following DTRF 150214 - 250N.m CPC20 after 24 hours

- When using grease prescribe grease with Molybdenum (do not prescribe oil or anticorrosive grease)
- Indicate if it is necessary to use thread lock and the type
- Authorization of the tightening head side: only in the case of a screwed assembly, in case of difficult
 access, specify the possibility of tight head side (by default the DTRF 150214 specifies the tightening
 nut side)
- Authorization to not mark the elements (blue, orange or green) according to the limits of the DTRF 150214: contact with effluent, difficulty of access, ...





6. MECHANICAL FASTENERS LIST

6.1. AUTHORIZED AND STANDARDIZED FASTENERS

PPLs (Preferred Part Lists) are built with these fasteners which are to be preferentially used. PPPLs (Project Preferred Parts Lists) are built with this PPLs and project specificities.

6.1.1 AUTHORIZED AND STANDARDIZED SCREWS



The authorization to use hexalobular (Torx) is managed through the PPL.

- (a) Authorized under condition of required towards fatigue holding and shear
- (b) For NAM market





Conventional	elements	Systematically replaced by
	Slotted pan head (CLS) screw	
1	Single-slotted truss head (RL S) screws 1ft	
	Double-slotted truss head (RL S) screws 2ft	
	CBHC Screw	
	Hexagonal socket head cap (CHC) screw, <u>low head</u>	or
TORX-PLUS	Torx plus screw (socket type or not)	
	Slotted cheese head (CS & SZ) screw	TORX
year a state and a	lotted flat countersunk head (F S) screw + conical washer	
	Slotted flat countersunk head (FS) screw	
	Cross recessed flat countersunk head (F Z) Philips or Pozidriv screw	
*	Socket flat countersunk head or bombed head screw for wood pozidrive or philipps	Screw for wood FX or BX, (hexalobular)
F 03-004 F 03-004 F 03-005 F 03-005	Notched screw	Tamper proof - Torx (countersunk with central plot)



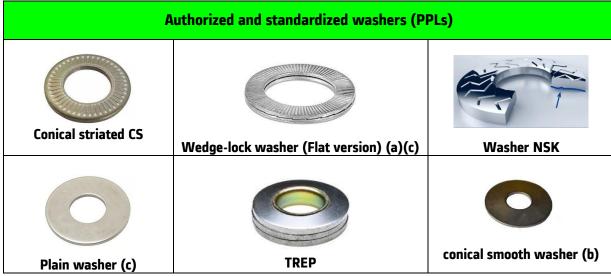


6.1.2 AUTHORIZED AND STANDARDIZED NUTS

Autho	orized and standardized nuts (P	PLs)
H (usual)	Locknut with plastic insert (c) H lock PLAST	Cage nut
Insert Keensert (a)	Crimping nut with an open hexagonal barrel (b)	Insert for wood

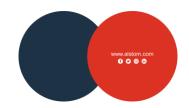
- (a) For structural application
- (b) For retrofit, use the broaching tool to mount hexagonal insert.
- (c) For mechanical application except for high temperature applications (and electrical application for d<=M5).

6.1.3 AUTHORIZED AND STANDARDIZED WASHERS



- (a) Preferentially on screwed assemblies and for bolting assemblies access required to the screw head side to hold it in position during tightening. To not associate a ramp washer with a flat washer that is not locked against rotation. The hardness of the material under the washer must be ≤ 200 HV.
- (b) For NAM market authorized under condition, for fatigue holding and shear
- (c) Does not work for settlement control (relaxation due to several layer, paint thickness, ...)





6.1.4 Non-standardized fasteners

To limit the number of references (rationalization), the authorization for implementing this type of element is only granted if:

- The quantity is justified
- Extension of existing solution
- Validation of the single-source aspect (if applicable).
- Agreement from maintenance (management of variants).

These fasteners are not included into the PPL and an authorization based on solid reason, can be given by local assembly expert to be included into a 3PL.

	Non standard elements	
Screw with integrated washer	Tab washer (a)	Hexagonal castellated nut – engaging or non engaging (b)
Hexagon bolts with flange (g)	H screw with flange or captive washer (conical or flat)	Tension nut
Thin nut class 04 (d)	conical smooth washer (c)	Ramp washer with internal plastic ring (k)
VS (f)	Verbus ripp screw or nut (e)	Cap nut





	Non standard elements	
Square head screw (c)	Wing nut	Thread cutting screw
Tension control bolt (hexagonal	Panhead screw (with 6 internal	Square collar screw
head)	lobes)	or carriage bolt
Screw FHC steel 8.8	Crimping nut with a closed hexagonal barrel	Steel welding nut (grade 06) or stainless steel
	V V S S	
Insert helicoïl (h)	VARGAL nut (i)	Locknut: Double-slotted (a)
Locknut: FS Nut (j)		

- (a) Against loss for electrical application only and d>M5
- (b) For historical application on component (like bogie): example conical smooth for assemblies with paper join
- (c) Authorized under condition of required towards fatigue holding and shear
- (d) Authorized for counter nut solution
- (e) Forbidden on soft surface or painted authorized for German projects
- (f) Preferentially for bolting assembly, with high shear load
- (g) In link with ramp washer (see chapter washer combination)
- (h) Limitation of use for fitting application or repair
- (i) Only for specific application with high vibration and validated by product qualification
- (j) Authorized for German projects
- (k) Authorized with technico-economic documentary evidence, industrial and logistic.





6.2. PROHIBITED FASTENERS

Fasteners listed in this section are not to be used on new projects.

This list was built on negative return of experience of these fasteners (not efficient fasteners or impacts on safety or maintenance) but also to reduce the number of fasteners solutions (rationalization).

6.2.1 PROHIBITED SCREWS



- a) Derogation possible based on a technico-economic documentary evidence, linked to quantities, industrial and logistic: Precoat links to high logistic constraints bounds to respect of best before dates and storage conditions
- b) Derogation possible based on a technico-economic documentary evidence, industrial and logistic. Due to high friction factor dispersion only to use after validating the performance class and tightening torque (product qualification)





6.2.2 PROHIBITED NUTS



- (a) Authorized when required by Project Contract (NAM) or for assemblies never dismount during maintenance.
- (b) Forbidden because need a precise drilling tolerance





6.2.3 PROHIBITED WASHERS

	Prohibited washers	
		The state of the s
Spring locks washer (c)	Onduflex washer	Fan washer (b)
Ripp lock washer	Ramp washer with plastic ring outside (d)	Wedge-lock washer - Conical version (a)
The Sur	TO THE STATE OF TH	
Internal tooth lock washer	Toothed lock washer	Cup washer

- (a) Example X serie from Nordlock Rationalization on flat wedge-lock washers only
- (b) Usable in dispensation, to respect the standard "hydraulic wedging for wheels" used by le Creusot
- (c) Authorized spring locks when required by Project Contract (NAM)
- (d) Forbidden due to rationalization reason and weakness against corrosion in highly exposed areas.





7. LIMITATIONS ON STANDARDIZED FASTENERS

7.1. LIMITATIONS ON DIAMETER AND MATERIAL

7.1.1 METRIC DIAMETERS



(*) historical diameter, always used on standard component like dampers.

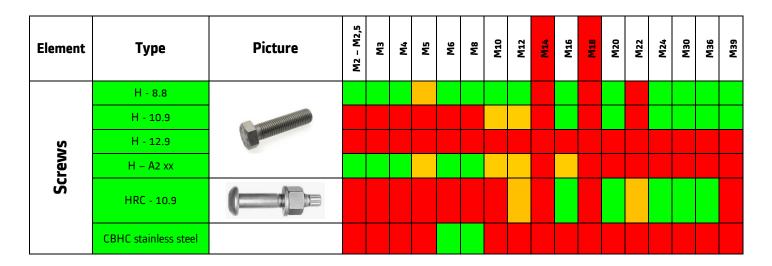
7.1.2 IMPERIAL DIAMETERS



(*) indicates non-structural applications - It is recommended to use 1/4" screws or above for equipment mounting or structural assemblies.

7.2. LIMITATIONS ON DIAMETER AND ELEMENT TYPES

7.2.1 METRIC DIAMETERS







Element	Туре	Picture	M2 - M2,5	Мз	M4	Ms	M6	M8	M10	M12	M14	M16	M18	M20	M22	M24	M30	M36	M39
	CBX StSt and 8.8 CBHC steel	(
	FHC Stainless steel	Or TORX																	
	FHC Steel FX StSt and 8.8	Or TORX																	
	CHC - 8.8																		
	CHC - 10.9																		
	CHC - 12.9	0																	
	CHC – Stainless steel	A																	
	Square neck bolt																		
	CBX, FX, Tamper Torx (countersunk with central plot) Only carbon steel																		
	HC(1)																		
	Screws for steel sheet or wood	W. Farther State of the State o																	
	Screw with integrated washer, captive																		
	With flange																		
	Thread cutting screw	*																	
	Thread forming screw																		





Element	Туре	Picture	M2 – M2,5	Мз	M4	Ms	M6	M8	M10	M12	M14	M16	M18	M20	M22	M24	M30	M36	M39
Insert	Helicoïl																		
lns	Keensert																		
	H nut																		
	Locknut double slotted																		
	FS																		
	Locknut with plastic insert																		
Nuts	Hexagonal blind rivet nut open (2)																		
Ž	Welded nut																		
	Cage nut																		
	Cap nut																		
	Wing nut																		
	Tension nut																		
Was	CS																		





Element	Туре	Picture	M2 - M2,5	M3	7W	M5	M6	M8	M10	M12	M14	M16	M18	M20	M22	M24	M30	M36	M39
	Flat (thick included)																		
	NSK																		
	TREP																		
	Wedge lock washer (flat)																		
	Wedge lock washer (flat) with elastic ring inside (3)																		
	VS	0																	

- (1) Carbon Steel Hexagon Socket Set Screws are not to be preloaded in tension.
- (2) Authorized for historical electrical application in M12 size (isolator) even it's red marked for rationalization on new project.
- (3) Only available on the market M8-M16.

7.2.2 IMPERIAL DIAMETERS

Element	Туре	Picture	7 #	9#	#8	#10	1/4"	5/16"	3/8"	7/16"	1/2"	9/16"	5/8"	3/4"	.,8/2	1"	1-1/8"	1-1/4"	1-3/8"	1-1/2"
	H – SAE Grade 5																			
	H – SAE Grade 8																			
	H – ASTM F593	<u> </u>																		
Screws	Machine Screw – No Grade FLZ CLZ RLZ	- H																		
S	HRC																			
	HSB stainless steel ASTM F879																			





Element	Туре	Picture	##	9#	8	#10	1/4"	5/16"	3/8″	7/16"	1/2"	9/16"	.8/9	3/4"	8/2	1"	1-1/8"	1-1/4"	1-3/8"	1-1/2"
	HSB Alloy Steel ASTM F835																			
	HSF Stainless steel ASTM F879	or Foxx																		
	HSF Alloy Steel ASTM F835																			
	HSC – Alloy Steel ASTM A574																			
	HSC – Stainless steel ASTM F837	0																		
	Square neck bolt																			
	HSBP																			
	HSFP																			
	HSS ⁽¹⁾																			
	Screws for steel sheet or wood	W. P. S. L.																		
	Screw with integrated washer, captive																			
	With flange																			
ert	Helicoïl																			
Insert	Keensert																			





Element	Туре	Picture	7#	9#	8#	#10	1/4"	.91/9	3/8″	7/16"	1/2"	9/16"	2/8,,	3/4"	8/2	1"	1-1/8"	1-1/4"	1-3/8"	1-1/2"
	Н																			
	H LOCK METAL																			
	H LOCK PLAST ⁽²⁾																			
	Hexagonal rivet nut Thin head, CS																			
	Hexagonal rivet nut Thin head, SS																			
50	Hexagonal rivet nut Flush head, CS																			
Nuts	Hexagonal rivet nut Flush head, SS																			
	Welded nut																			
	Cage nut, CS																			
	Cage Nut, SS																			
	Acorn nut																			
	Wing nut																			
	Tension nut																			
Washers	C washer																			
Was	PLAIN																			





Element	Туре	Picture	7#	9#	#8	#10	1/4"	5/16"	3/8"	7/16"	1/2"	9/16"	5/8"	3/4"	.,8/2	1"	1-1/8"	1-1/4"	1-3/8"	1-1/2"
	HRDN																			
	RAMP																			
	SPRING ⁽³⁾																			

- (1) Carbon Steel Hexagon Socket Set Screws are not to be preloaded in tension.
- (2) When zinc flake plating is required, Carbon Steel Locknuts with Plastic Inserts shall not have plastic inserts installed during the plating process.
- (3) Use spring locks when required by Project Contract.

7.3. LIMITATIONS ON SCREW LENGTHS

Limitation for screws lengths is to be respected according to the PPLs. Regarding length thread of screw, fully threaded screws to be preferentially used except if stress under fatigue requires the use of partially threaded screws or long assemblies (example: $L_k>5d$).

7.4. LIMITATIONS ON FASTENERS DEPENDING ON REGIONS

Restrictions on element types depending on areas are to be respected according to PPLs.





8. DESIGN RULES

8.1. SELECTION AND ASSOCIATION RULES

8.1.1 SELECTION ACCORDING ZONES

		Zones	
Diameter	Z _{inD} (Inside dry area)	Z _{inW} (Interior wet area)	Z _{out} (outside)
≤ M6		Stainless steel or carbo	n steel if galvanic
or	Stainless steel	incompatib	•
≤ 1/4"		incompatit	Jility
≥ M8	Carbon steel	ool*	
or ≥ 5/16"	Carbon Steer	Carbon st	cci

^(*) Additional protection, decoupling if issues with galvanic compatibility with the assembled materials.

8.1.2 Associations rules following mechanical characteristics

Metric units

Screw grade	Washer minimum hardness *	Nut quality class		
8.8	200 HV	8		
10.9	300 HV	10		
12.9	420 HV	12		
A2 70	200 HV	A2 70**		
A4 80	200 HV	A4 80		

^(*) as per DTRF 150213

Imperial units

Screw grade	Washer	Nut grade
SAE J429 Grade 5	ASME B18.21.1	SAE J995 Grade 5
SAE J429 Grade 8	ASTM F436	SAE J995 Grade 8
ASTM F593 CW	ASME B18.21.1	ASTM F594 CW

In the design, it is prohibited to:

 Mix screws or nuts of different quality classes and washer hardness's on output from engineering office.

^(**) or some HFR nut in A4-80 (with the Torque value for 70 quality class in line with the screw)





8.1.3 Association rules about washer

Combinations of washers to use versus the holes size (mechanicals applications):

	Round (R)		Steel support ** Size of clearance hole				
	or oblong	Part treatment	F or M	L			
	(O) hole		Close or Normal Fit	Loose Fit			
Plain washer	R	Painted or unpainted	Regular Plain				
NSK	R or O	part or with ASP	NSK-M	NSK-M + large flat washer*			
		Unpainted part	Larg	e CS			
	R	Painted part or with ASP	Large CS				
CS	0	Painted or unpainted part or with ASP	+ large flat washer*				
VS	R	Unpainted part or with ASP	VS				
	R	Painted or unpainted	Regular Plain+ Spring	Wide Plain + Spring			
Spring lock washer***	0	part or with ASP	Regulal Flam+ Spring				
+		Unpainted	С				
	R	Painted part or with ASP	C + regular plain	C + wide plain			
c	0	Painted or unpainted part or with ASP	washer	washer*			
	R		Ramps washers large series				
Wedge lock washer - large series	0	Unpainted part or with ASP	Forbidden ***				

ASP = Assembly Surface Protection

(*): if the stiffness of the large flat washer is insufficient, or if problems arise when purchasing washers following standard (thick washer), it is possible to insert a special washer under the wide CS, as per the drawing with a hardness compatible with the screw quality class and whose thickness \geq 0.5 d.

(**) on soft material, such as aluminum, take a wide washer (wide CS or wedge lock washer).

For mechanical use only (not electrical) note that thick flat washers are not used, since they are difficult to source.

(***) solution for previous projects: Large ramps lock washer + screw with flange

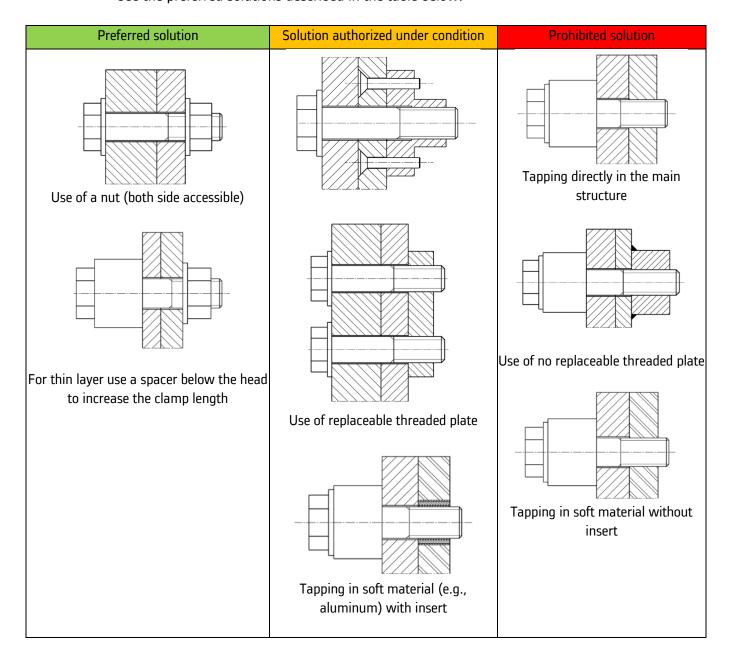
(****) Use spring locks when required by Project Contract.





8.1.4 DESIGN RULES

- Use for a single fixing point where several screws or bolts are used (screw and nut), the same diameter and the same quality class for all elements at the fixing point.
- For adjustment areas, which require re-assembly, in order to avoid replacement of fasteners, it is requested to use flat washers and H-nuts which can be re-used according to the table of re-use of fasteners in DTRF 150214.
- Use the preferred solutions described in the table below:







Preferred solution	Solution authorized under condition	Prohibited solution
Use of a nut blocked in rotation (one side accessible only for tightening)	Spacer not blocked into rotation, on the nut side if validated by indus	Tapping between two parts Welded nut or locknut
Screw countersunk head with nut		Screw countersunk head with rivet nut, cage nut or welded nut
		Screw flat end or socket welded into thin sheet





Preferred solution	Solution authorized under condition	Prohibited solution
	1/- X	
Stud properly implemented	Implementation with adjustment (e.g., risk of collision)	Stud bottoming out in threaded hole

Protruding thread length (b), unused thread length (c) and edge distance (e) must respect following limits, exceptions permitted for economic, availability (per PPL) or clearance reasons.

 $b \ge 3p$ for Lkmax,

b ≤ 10p,

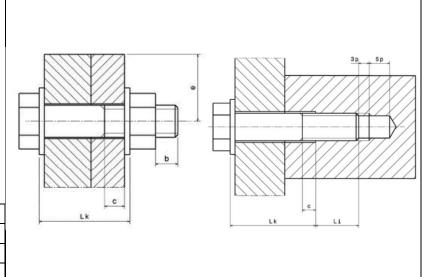
c ≥ 6p,

e ≥ 1.5d

Li validated by calculation team.

Lkmin Clamp length:

	Lk min
Structural assembly	>3d
Semi structural assembly	>1d
Fastening assembly	>0,5d







Thread-lock

- Thread-lock is used to prevent loss and not to prevent loosening.
- The design should be made to avoid the use of thread-locker as much as possible.
- If this cannot be avoided, the product is the medium thread-locker, applicable for all diameters and only according to the following cases:

Disassembly	Settlement	Function against loss	Nut typ	Washer typ	Stainless steel fasteners	Non-stainless steel fasteners		
Bolted or screwed assembly frequently disassembled during maintenance								
	with re- tightening				Prohibited			
		no				Not necessary		
Bolted or screwed assembly not	ably not		Tapping, H nut, crimp nut with hexagonal	wedge-lock washer or collapsible washer	Not	Not necessary		
disassembled during maintenance	tightening	yes	barrel	Flat washer, CS, VS	permitted*	Thread-lock		
			Locknut, cage nut, insert (Keensert or Helicoil)	any washer		Prohibited		

^(*) except for the fitting of small stainless-steel screws tightened without lubrication - DTRF150214

8.2. FASTENER ADDITIONAL PROTECTION AFTER ASSEMBLY

- If the assembly is visible with aesthetic requirements, it is mandatory to adapt the fastener type (color, material, geometry, ...)
- If the assembly is visible without any aesthetic requirement, it is necessary to protect fastener bought following DTRF 150213 from corrosion with an additional protection following the below table (formalization following DTRF150250 DADI). The reference of the proper additional protection (in accordance with surface protection catalogue DTRF150611), depending on the constraints of the manufacturing process is defined by the industrialization in instruction sheets.





				Zones						
					Z _{inD} Z _{inW}				Z _{out} *	
					Exposure	to cleanin	g agents a	ccording to	DTRF 15026	52
Material	Family DTRF 150213	Disassembly frequency	Visible assembly without aesthetic requirement	Unexposed	C1 or C2 diluted	C3 or C2 non diluted	Unexposed	C1 or C2 diluted	C3 or C2 non diluted	All exposures
Stainless steel	C/D	D / ND	V / NV	0	0	0	0	0	0	0
		D / ND	NV	0	0	2	0	0	2	2
7:	Α	D	V	0	1	1	0	1	1	1
Zinc-		ND	V	0	2	2	0	2	2	2
platted steel		D / ND	NV	0	2	2	2	2	2	2
Steel	B / D**	D	V	0	1	1	1	1	1	1
		ND	V	0	2	2	2	2	2	2

(*) with or without exposure on UV radiation

(**) D includes pins and studs

D: Disassembled frequently in maintenance

ND: Not disassembled frequently in maintenance (at midlife or never)

V: Visible assembly NV: Assembly not visible

0	Additional protection not required
1	Additional protection not permanent (wax,)
2	Permanent additional protection (varnish, paint,)





9. FUNDAMENTAL DIMENSIONING CRITERIA

9.1. ASSEMBLY DIMENSIONING

Utilization constraints	Criterion	Level	Flexibility
Resistance under axial load: exploitation (exceptional and service) and proof load	Axial load	No separation	FO
Resistance under transversal load in service conditions	Shear load	No sliding	FO
Resistance under transversal load in proof load conditions	Shear load	No sliding*	F1
	No shearing of the dowel	Under Proof Load	Fo
Dimensioning of the mechanical stop (example of a pin if the screws used cannot absorb the shear load)	Tolerance at shearing dowel level	Enough to ensure non- contact with the fastener shaft	FO
	No peening of the dowel or the boring	Under Proof Load	FO
Shear stress directly absorbed by a screw		Prohibited	FO
All the criteria must be inside of the elastic domain:			
Utilization rate on tightened bolt at installation stage	Yield	90% +/- 3%	Fo
Utilization rate on tightened bolt during exploitation	Yield	100%	FO

- (*) In case of exceptional loads sliding may be accepted if the sliding distance is limited:
 - By a transition fit for load transferring interfaces (e.g. pivot, articulation)
 - o tolerance of form fit < limit displacement
 - o proof of bending stress in bolt / fit tolerance
 - o proof of form fit (see dimensioning of the mechanical stop)
 - o proof of full functionality retained
 - locking device against self-loosening by rotation
 - By a loose fit for e.g. equipment fixation or one directional loads (e.g. steps, slinging hook)
 - proof of shear resistance and bearing
 - o proof of form fit
 - o limit displacement by hole size (m or f) or the bolt resilience





Design criterion for mechanical screwed assemblies inside an electrical box of equipment (traction block or box, LV cabinet, switch gear, etc.) whose safety level is:

- High: dimensioning is performed by following the instructions of the previous table.
- Medium or low: dimensioning may be performed according to EN 1993-8 (calculation based on the shear failure limit of the screw)

Design criterion for an anti-fall device:

- Primary fastener according to the requirements of the previous table
- Fastening of anti-fall device or second loading path, may be performed with the following criteria:
 - Separation and slippage possible
 - No rupture of the fastener.

9.2. OPTIMIZATION RULES FOR BOLTED ASSEMBLIES

By order of priority:

- 1. Increase the clamping length
- 2. Increase number of screws in order to decrease the preload required into each screw
- 3. Increase screw diameter in order to increase minimum preload F0min
- 4. Increase bolt grade in order to increase minimum preload F0min
- 5. Increase shear force capacity using improved friction factor at interface
- 6. Increase performance class (CP) in order to increase minimum preload F0min
- 7. Optimize the torque value (by calculation or test)*
- 8. Specify retightening operation
- 9. Lubricate the screw or the nut in order to increase minimum preload with manufacturing validation (in that case, contact the Site Assembly Expert).
- (*) About torque value, the rule is always to use the standard tightening torques indicated in DTRF 150214. Only in the following cases, can the assembly be optimized (tension reached) by a specific torque value (nonstandard value). Please note that this specific torque must be as close as possible to the rounding table of the standard tightening means given in DTRF 150214 (otherwise industrial validation is required):
 - Use 90% of the yield point if standard Torque value is using a below value for standardization purpose
 - Assembly having been the subject of a product qualification considering serial part (rigidity of the real
 assembly, bearing limitation, ...) or special elements (hammer screw, washer or nut) different from
 the preferred part list.
 - Tightening on the head side (ex: case of screws with flange) and not the nut side.
 - Use of washers or nuts different from the reference tables of the DTRF 150214
 - Tightening into a soft thread (specific friction factor or limitation related to the type of insert)
 - Tightening on non-metallic part (plastic, composite, ...)





10. DEPARTMENT RESPONSIBILITY AND TIGHTENING METHOD

The engineering department is responsible for determining the intensity of the efforts exerted on the assembly, for dimensioning the assembly and for defining fastening elements designation, as well as the minimum, nominal and maximum pre-stresses.

In cooperation with the methods office, depending on tightening tools available in production and maintenance, their accessibility, their required precision, a tightening method used to reach the nominal tension is selected (tightening method - torque, tensioning, fusible bolt, tightening path ...). The engineering office will validate the selected tightening method and will then indicate the selected performance class.

The methods department will then determine the tightening tool adapted to the performance class that will be used for the assembly and bolting sequences.

11. MOUNTING PRESCRIPTIONS

The mountings prescriptions, standard tightening torque, inspections to be made before and after tightening, control method, retightening, re-using of components are indicated into ALSTOM Standard DTRF 150214.